# Mathematical Modeling <br> Project fall 2016 

## Diesel Particulate Filter



## Background

- Loss of injectivity in polymer enhanced oil recovery



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- Mechanically trapped polymer junk
- Master projects at SINTEF


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- Master projects at SINTEF
- Volkswagen emissions scandal



## Filter design

- Diesel motor emits soot particles


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- Heath hazard
- Exhaust gas enters honeycomb structure and it forces to flow through a porous media



## Filter efficiency

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- List up parameters: pressure drop, $u_{\text {in }}, u_{\text {out }}$, Permeability $K$...
- Q: Identify the filter design parameters.


## Modeling equations

- Conservation laws
- Conservation of mass
- Conservation of momentum
- Conservation of energy


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$$
\begin{gathered}
\rho_{t}+\nabla \cdot(\rho u)=0 \\
\left\{\begin{array}{c}
\text { rate of } \\
\text { change }
\end{array}\right\}=\left\{\begin{array}{c}
\text { what } \\
\text { comes } \\
\text { in }
\end{array}\right\}-\left\{\begin{array}{c}
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$$

- Hence,

$$
\frac{d}{d t} \int_{\Omega} \rho d x=-\int_{\partial \Omega} \rho u \cdot \boldsymbol{n} d x
$$

implies

$$
\int_{\Omega} \frac{\partial \rho}{\partial t} d x=-\int_{\Omega} \nabla \cdot(\rho u) d x
$$

## Conservation of momentum



## Conservation of momentum


momentum $=m \boldsymbol{v}$

$$
\frac{d}{d t}(m \boldsymbol{v})=\boldsymbol{f}
$$

## Conservation of momentum

- Conservation of momentum for a fluid

$$
(\rho u)_{t}+\nabla \cdot(\rho u \otimes u)=f
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Q: Derive this equation starting from Newton's law.

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- Here, $f$ is an external volumetric force and

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\begin{gathered}
u \otimes u=\left(\begin{array}{lll}
u_{1} u_{1} & u_{2} u_{1} & u_{3} u_{1} \\
u_{1} u_{2} & u_{2} u_{2} & u_{3} u_{2} \\
u_{1} u_{3} & u_{2} u_{3} & u_{3} u_{3}
\end{array}\right), \\
\nabla \cdot(\rho u \otimes u)=\left(\begin{array}{l}
\frac{\partial}{\partial x_{1}}\left(\rho u_{1} u_{1}\right)+\frac{\partial}{\partial x_{2}}\left(\rho u_{1} u_{2}\right)+\frac{\partial}{\partial x_{3}}\left(\rho u_{1} u_{3}\right) \\
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- Divergence theorem for matrices

$$
\int_{\Omega} \nabla \cdot A(x) d x=\int_{\partial \Omega} A(x) \boldsymbol{n} d x
$$

## Newton and Lagrange

- Introduce kinetic and potential energy

$$
E_{\mathrm{kin}}=\frac{1}{2} m \dot{x}^{2} \quad \text { and } \quad E_{\mathrm{pot}}(x)
$$


$m \ddot{x}=f$

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\mathcal{L}=\int_{0}^{T}\left(E_{\mathrm{kin}}-E_{\mathrm{pot}}\right) d t
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\delta \mathcal{L} & =\int_{0}^{T} m x \cdot \delta \dot{x} d t-\int_{0}^{T} \nabla E_{\mathrm{pot}}(x) \cdot \delta x d t \\
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- Hence,

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m \ddot{x}=\nabla E_{\text {kin }}
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f_{\text {diss }}=\lambda(\nabla \cdot u) \boldsymbol{n}+\mu\left(\nabla u+\nabla(u)^{T}\right) \boldsymbol{n},
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$$
u_{1}(x, y)=v y, \quad u_{2}(x, y)=0
$$

$$
\nabla u=\left(\begin{array}{ll}
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0 & 0
\end{array}\right), \quad \nabla \cdot u=\operatorname{tr}(\nabla u), \quad \boldsymbol{n}=\binom{0}{-1}
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$$
f_{\mathrm{diss}}=\mu\left(\begin{array}{ll}
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- Viscous energy

$$
E_{\mathrm{diss}}=\int_{\Omega}\left(\frac{\lambda}{2} \operatorname{tr}(\varepsilon)^{2}+\mu\|\varepsilon\|^{2}\right) d x
$$

where $\varepsilon$ is symmetric tensor

$$
\varepsilon=\frac{1}{2}\left(\nabla u+\nabla(u)^{T}\right)
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## Conservation of momentum and angular

## momentum

- Newton's third law

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- For a closed system, we have the conservation of momentum and of angular momentum

$$
\frac{d}{d t} \sum_{i}\left(m_{i} v_{i}\right)=0 \quad \text { and } \quad \frac{d}{d t} \sum_{i} m_{i} r_{i} \times v_{i}=0
$$

## Approximation of the viscous force

- Infinite channel in a stationary state:
$\xrightarrow{\bar{u}} \underset{\Delta \bar{p}}{\stackrel{\rightharpoonup}{\rightrightarrows}} \stackrel{\Omega}{\rightrightarrows}$


## Approximation of the viscous force

- Infinite channel in a stationary state:

- Define average values over the cross-section,

$$
\bar{u}=\frac{1}{|\Omega|} \int_{\Omega} u d x \quad \text { and } \quad \bar{p}=\frac{1}{|\Omega|} \int_{\Omega} p d x
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Q: Express $f_{\text {diss }}$ as a function of $\bar{u}$. Show that $f_{\text {diss }}=-a \bar{u}$, for a constant a that depends only on the shape $\Omega$. You have to compute the velocity profile.

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- Similar approach to obtain Darcy approximation $\bar{u}=-K \frac{\Delta \bar{p}}{\Delta x}$.
- Q: Compute the coefficient a for $\Omega$ given by a cylinder, a square or an hexagon.


## One dimensional approximation

Instead of considering

$$
\begin{aligned}
\rho_{t}+\nabla \cdot(\rho u) & =0 \\
(\rho u)_{t}+\nabla \cdot(\rho u \otimes u) & =-\nabla p-a u
\end{aligned}
$$

we consider the one dimensional approximation

$$
\begin{aligned}
\rho_{t}+(\rho u)_{x} & =0 \\
(\rho u)_{t}+\left(\rho u^{2}\right)_{x} & =-\frac{\partial p}{\partial x}-a u
\end{aligned}
$$

Q: How are the one-dimensional equations obtained? What are the approximations that we are doing?

## One dimensional approximation (2)

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p=b \rho
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- Q: Set up equations for the time-dependent perturbed solution from the steady-state.

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- After that, we only consider steady-state.


## mass loss from the walls

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- Q: How are the $1 D$ equations changed?


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- Q: How are the full 3D equations changed?
- Q: How are the 1D equations changed?
- Q: Solve numerically the 1D equations in the stationary case.


## Porous media

- Linear relation between velocity and pressure drop

$$
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The scalar (or matrix) $K$ is called the permeability.


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- The Darcy relation replaces the momentum equation. Note that the kinetic energy is neglected in a porous media.
- Q: Derive the governing equation for a porous media.



## Upscaling of the porous media layer

- We want to avoid solving the partial differential equations the porous media layer.


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- Let $p_{i}$ and $p_{e}$ be the internal and external pressure, and

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U=\int_{\Gamma} u \cdot \boldsymbol{n} d x
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Cylindrical channel


Square channel

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U=\int_{\Gamma} u \cdot \boldsymbol{n} d x
$$



Cylindrical channel


Square channel

- Q: Find the relation between $U$ and $p_{e}-p_{i}$ for the steady state. Compute $\kappa$ for a cylinder and a square.

$$
U=\kappa\left(p_{e}-p_{i}\right)
$$

## Coupling of the channel with a porous layer

- At the interface, we should have conservation of mass and force balance. Q: What are the interface condition?



## Coupling of the channel with a porous layer

- At the interface, we should have conservation of mass and force balance. Q: What are the interface condition?

- We consider pressure continuity at the interface (consistent with previous approximation). $\mathbf{Q}$ : Set up the one dimensional equations.



## Coupling of the channel with a porous layer (2)

- In the stationary case, the equations take the form

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- Q: What kind of input/output do we want to consider?
- Q: Solve the equations numerically.


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$u_{\text {in }}, p_{\text {in }} \rightarrow \quad(u, p)$
- Q: Setup the equations for the one dimensional model in the stationary case.

$$
A(p, u, q, v)\left(\begin{array}{c}
p_{x} \\
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v_{x}
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## Accumulation of particles

- We denote by $c$ the concentration of soot particles $(c=$ mass(soot)/mass(gas))



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- Q: Compute this value numerically (one-dimensional case).


## Filter clogging

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- The coefficient $\kappa$ depends on the amount of accumulated soot and the amount of accumulated soot depends on $\kappa$. Q: Set up the coupled equations.


## Filter clogging - general cross-section

- For a general geometry of the cross-section, the particles do not accumulate uniformly at the interface

with

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- Q: Derive the equation for $x(\tau, t)$. Find an equation that guarantees the conservation of the mass of soot.
- Q: Design and implement a scheme to solve the governing equations for this problem.

