Mathematical Modeling Project fall 2016

Diesel Particulate Filter



Loss of injectivity in polymer enhanced oil recovery



► Loss of injectivity in polymer enhanced oil recovery



Master projects at SINTEF

Loss of injectivity in polymer enhanced oil recovery



- Master projects at SINTEF
- Volkswagen emissions scandal

VAN DAM



exhaust scandal: faked emissions test

Diesel motor emits soot particles

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- Exhaust gas enters honeycomb structure and it forces to flow through a porous media



Gereinigtes Abgas ohne Ruß Treated exhaust gas without particulates Amount of particles that are filtered versus energy that is used to operate the filter.

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Energy balance



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 - **Q**: Identify the filter design parameters.

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$$\bullet \text{ Hence,}$$

$$\frac{d}{dt} \int_{\Omega} \rho \, dx = - \int_{\partial \Omega} \rho u \cdot \boldsymbol{n} \, dx$$

$$\inf \text{ implies}$$

$$\int_{\Omega} \frac{\partial \rho}{\partial t} \, dx = - \int_{\Omega} \nabla \cdot (\rho u) \, dx$$





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$$u \otimes u = \begin{pmatrix} u_1 u_1 & u_2 u_1 & u_3 u_1 \\ u_1 u_2 & u_2 u_2 & u_3 u_2 \\ u_1 u_3 & u_2 u_3 & u_3 u_3 \end{pmatrix},$$

$$\nabla \cdot \left(\rho u \otimes u\right) = \begin{pmatrix} \frac{\partial}{\partial x_1} (\rho u_1 u_1) + \frac{\partial}{\partial x_2} (\rho u_1 u_2) + \frac{\partial}{\partial x_3} (\rho u_1 u_3) \\ \frac{\partial}{\partial x_1} (\rho u_2 u_1) + \frac{\partial}{\partial x_2} (\rho u_2 u_2) + \frac{\partial}{\partial x_3} (\rho u_2 u_3) \\ \frac{\partial}{\partial x_1} (\rho u_3 u_1) + \frac{\partial}{\partial x_2} (\rho u_3 u_2) + \frac{\partial}{\partial x_3} (\rho u_3 u_3) \end{pmatrix}$$

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Divergence theorem for matrices

$$\int_{\Omega} \nabla \cdot A(x) \, dx = \int_{\partial \Omega} A(x) \boldsymbol{n} \, dx$$



$$E_{\rm kin} = \frac{1}{2}m\dot{x}^2$$
 and $E_{\rm pot}$



$$m\ddot{x} = f$$



Introduce kinetic and potential energy

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 and $E_{\mathsf{pot}}(x)$

Set up the Lagrangian

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We have

$$\begin{split} \delta \mathcal{L} &= \int_0^T mx \cdot \delta \dot{x} \, dt - \int_0^T \nabla E_{\mathsf{pot}}(x) \cdot \delta x \, dt \\ &= \int_0^T \left(-m \ddot{x} + \nabla E_{\mathsf{pot}}(x) \right) \cdot \delta x \, dt \end{split}$$



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$$\delta \mathcal{L} = \int_0^T mx \cdot \delta \dot{x} \, dt - \int_0^T \nabla E_{pot}(x) \cdot \delta x \, dt$$

$$= \int_0^T (-m\ddot{x} + \nabla E_{pot}(x)) \cdot \delta x \, dt$$

► Hence,

$$m\ddot{x} = \nabla E_{\mathsf{kin}}$$

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$$\nabla u = \begin{pmatrix} 0 & v \\ 0 & 0 \end{pmatrix}, \quad \nabla \cdot u = \operatorname{tr}(\nabla u), \quad \boldsymbol{n} = \begin{pmatrix} 0 \\ -1 \end{pmatrix}$$
$$f_{\mathsf{diss}} = \mu \begin{pmatrix} 0 & v \\ v & 0 \end{pmatrix} \begin{pmatrix} 0 \\ -1 \end{pmatrix} = -\mu v \begin{pmatrix} 1 \\ 0 \end{pmatrix}$$

• If
$$u = \text{constant}$$
, then $f_{\text{diss}} = 0$

- \blacktriangleright If $u={\rm constant},$ then $f_{\rm diss}=0$
- Let us consider the velocity field u of rotational motion, we can prove that

$$u(t,x) = A(t)x,$$

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$$f_{\mathsf{diss}} = \lambda \operatorname{tr}(A)\boldsymbol{n} + \mu(A + A^T)\boldsymbol{n}$$
$$= 0.$$

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Viscous energy

$$E_{\mathsf{diss}} = \int_{\Omega} \left(\frac{\lambda}{2} \operatorname{tr}(\boldsymbol{\varepsilon})^2 + \mu \left\|\boldsymbol{\varepsilon}\right\|^2\right) dx$$

where ε is symmetric tensor

$$\boldsymbol{\varepsilon} = \frac{1}{2} (\nabla u + \nabla (u)^T).$$

Conservation of momentum and angular momentum

Newton's third law

When one body exerts a force on a second body, the second body simultaneously exerts a force equal in magnitude and opposite in direction on the first body



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 For a closed system, we have the conservation of momentum and of angular momentum

$$\frac{d}{dt}\sum_{i}(m_{i}v_{i})=0$$
 and $\frac{d}{dt}\sum_{i}m_{i}r_{i}\times v_{i}=0$

Infinite channel in a stationary state:



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Define average values over the cross-section,

$$\bar{u} = \frac{1}{|\Omega|} \int_{\Omega} u \, dx \quad \text{ and } \quad \bar{p} = \frac{1}{|\Omega|} \int_{\Omega} p \, dx$$

Q: Express f_{diss} as a function of \bar{u} . Show that $f_{diss} = -a\bar{u}$, for a constant a that depends only on the shape Ω . You have to compute the velocity profile.

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Q: Express f_{diss} as a function of ū. Show that f_{diss} = -aū, for a constant a that depends only on the shape Ω. You have to compute the velocity profile.
Similar approach to obtain Darcy approximation ū = -K Δp/Δx.

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- Similar approach to obtain Darcy approximation $\bar{u} = -K \frac{\Delta \bar{p}}{\Delta x}$.
- Q: Compute the coefficient a for Ω given by a cylinder, a square or an hexagon.

Instead of considering

$$\rho_t + \nabla \cdot (\rho u) = 0,$$

$$(\rho u)_t + \nabla \cdot (\rho u \otimes u) = -\nabla p - au,$$

we consider the one dimensional approximation

$$\rho_t + (\rho u)_x = 0,$$

$$(\rho u)_t + (\rho u^2)_x = -\frac{\partial p}{\partial x} - au.$$

Q: How are the one-dimensional equations obtained? What are the approximations that we are doing?

 We consider an ideal gas law, with constant temperature. Then p is proportional to ρ,

$$p = b\rho$$

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After that, we only consider steady-state.

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- ► **Q**: How are the full 3D equations changed?
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- ► **Q**: Solve numerically the 1D equations in the stationary case.

 Linear relation between velocity and pressure drop

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- Q: Derive the governing equation for a porous media.



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Cylindrical channel



Square channel

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Cylindrical channel

Square channel

• **Q**: Find the relation between U and $p_e - p_i$ for the steady state. Compute κ for a cylinder and a square.

$$U = \kappa (p_e - p_i)$$

Coupling of the channel with a porous layer

► At the interface, we should have conservation of mass and force balance. Q: What are the interface condition?



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Porous material (ρ_p, p_p, u_p) Ω_p Channel interior (ρ_d, p_d, u_d) Ω_d

 We consider pressure continuity at the interface (consistent with previous approximation). Q: Set up the one dimensional equations.



Coupling of the channel with a porous layer (2)

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- ▶ **Q**: Solve the equations numerically.

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- ► We introduce the pressure *q* and velocity *v* in the outlet channel.

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► **Q**: Setup the equations for the one dimensional model in the stationary case.

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- ► **Q**: What is the modeling equation that governs *c*, for the 3D model and the one-dimensional model.
- ► **Q**: Find a formula for the amount of accumulated particles as a function of t and x
- ► **Q**: Compute this value numerically (one-dimensional case).

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The coefficient κ depends on the amount of accumulated soot and the amount of accumulated soot depends on κ.
 Q: Set up the coupled equations.

Filter clogging - general cross-section

 For a general geometry of the cross-section, the particles do not accumulate uniformly at the interface



with

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- **Q**: Derive the equation for $x(\tau, t)$. Find an equation that guarantees the conservation of the mass of soot.
- Q: Design and implement a scheme to solve the governing equations for this problem.